PREMIXED FLAME PROPAGATION BETWEEN TWO CLOSELY SPACED PARALLEL PLATES

D. Fernández-Galisteo^{*a*}, J. Gross^{*b*}, and P.D. Ronney^{*b*}

^aDepartment of Energy, CIEMAT, Spain

^bDepartment of Aerospace and Mechanical Engineering, University of Southern California, Los Angeles, CA, USA

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Introduction

The propagation of slow quasi-isobaric premixed flames between two closely spaced and adiabatic plates is studied. The results can be of particular interest in microscale combustion where the confined flow modifies the intrinsic flame instabilities. For example, the viscosity contrast across the flame (Saffman-Taylor instability) becomes non-negligible for sufficiently close plates [1, 2]. We are interested to show a simple formulation that allows the study of flame insta-

Case 1 : Le=1, β=10, γ=5, G=0, σ=0, k=0



Fig. 2 depicts four cases, where the effects of bouyancy, viscosity contrast and differential diffusion on the flame stability have been isolated. The only effect included in case 1 as a baseline is thermal expansion. After a small period of acceleration due to the hot-spot initial conditions, case 1 and 2 show a 4-wave mode that propagate steadily before the occurrence of cell splitting and merging. Case 3 and 4 show a 2-wave mode propagation, although smaller wrinkled structures appear into the longest wrinkles in case 4 caused by the diffusive-thermal instability, as seen experimentally [1]. These structures modifies the global flame



front speed, which is plotted in Fig. 3. The front speed (S_T/S_L) is defined as

$$S_T/S_L = \frac{\iint_{xy} \omega \, \mathrm{d}x \, \mathrm{d}y}{L_y},$$

being L_y is the length in the y component.

The baseline thermal-expansion case 1 shows $S_T/S_T \approx 1.5$. This value is enhanced by 30% when the viscosity contrast is included, showing an important effect in very confined flows. Downward case 2 is slower than upward case, but still has velocities larger than one. Downward case 2 does stabilize the wrinkles in case 1 and slow the front velocity. The Low *Le* case 4 has a drastic effect on the front speed due to the emergence of the small cellular-like wrinkled structures. In all cases cell splitting and merging appears in the computations after a period of steady propagation.



 $F + O \rightarrow P$ and constant specific heat c_p , and becomes

$$\rho \frac{\partial \theta}{\partial t} + \rho U_x \frac{\partial \theta}{\partial x} + \rho U_y \frac{\partial \theta}{\partial y} = \nabla \cdot (\mu \nabla \theta) + \omega, \qquad (1)$$
$$\rho \frac{\partial Y}{\partial t} + \rho U_x \frac{\partial Y}{\partial x} + \rho U_y \frac{\partial Y}{\partial y} = \frac{1}{Le} \nabla \cdot (\mu \nabla Y) - \omega, \qquad (2)$$

where

$$\omega = \frac{\beta^2}{2s_L^2 Le} \frac{(1+\gamma)^{2-\sigma}}{(1+\gamma\theta)^2} Y \exp\left\{\frac{\beta(\theta-1)}{1+[\gamma/(1+\gamma)](\theta-1)}\right\},$$

together with the reduced Darcy's law for the z-averaged velocity

$$U_x(x,y;t)\,\vec{e}_x + U_y(x,y;t)\,\vec{e}_y = -\frac{\nabla p - \rho G \vec{e}_x}{\mu},\quad(3)$$

the pressure deviation field from the ambient

$$\Delta p = (1 + \sigma) G \frac{\partial \rho}{\partial x} - \frac{\sigma}{\rho} \nabla \rho \cdot \nabla p - \gamma \mu \left[\nabla \cdot (\mu \nabla \theta) + \omega \right], \quad (4)$$

and the equation of state

$$\rho = 1/(1 + \gamma \theta). \tag{5}$$

The following parameters appear: $\beta = E(T_a - T_u)/RT_a^2$, $\gamma = (T_a - T_u)/T_u$, $Le = \mathcal{D}_T/\mathcal{D}_F$, the buoyancy effect G, the temperature-dependent viscosity ratio in the form $\mu = \mu'/\mu_u = (1 + \gamma\theta)^{\sigma}$ and the reduced planar flame speed $s_L = S_L/(S_L)_{asp}$, where $(S_L)_{asp} = \sqrt{2Le\mathcal{B}\lambda_b}/(\beta^2c_p) (\rho_b/\rho_u) \exp(-E/2RT_a)$. The boundary conditions for the reduced 2D problem (1)-(5) correspond with periodic conditions at y_{min} and y_{max} , together with p = 0 at x_{min} and $U_x = 0$ at x_{max} .

References

 [1] J. Gross, X. Pan, P. D. Ronney. Spring Technical Meeting, Combustion Institute, Western States, Section, March 24-25, Pasadena, CA.

Results

Time-dependent computations were carried out in a domain large enough to capture the wrinkled flame structures correctly. The initial condition was chosen in the form of three hot spot at (x, y) = (0, 0), (0, 50), and (0, 100).

Fig. 2: Flame front (given by isotherms contour) and flow velocity vectors calculated for four differents cases. [2] S. H. Kang, H. G Im, S. W. Baek. Combust. Theory Modelling 7 (2003) 343-363.

[3] V. N. Kurdyumov, M. Matalon. Proc. Combust. Inst. 34 (2013) 865-872.



